

Low-frequency vibrations of strongly inhomogeneous layered elastic plates

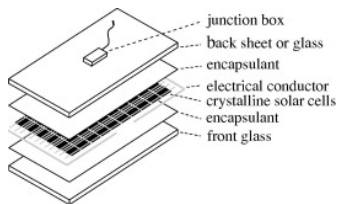
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- Introduction with practical motivation
- In-plane vector problem
 - Multi-parametric analysis of a dispersion relation
 - Two-mode approximation
 - Sketch of asymptotic PDEs
- Out-of-plane (anti-plane) scalar problem
 - Multi-parametric analysis of a dispersion relation
 - PDEs for the lowest shear mode
- Out-of-plane problem for asymmetric plate
 - Two-mode approximation

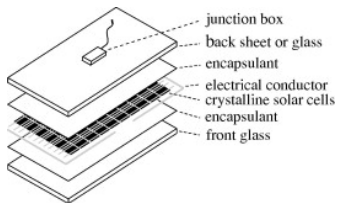
High-contrast layered structures

- Photovoltaic panels ($\mu_c/\mu_s \sim 10^{-2} - 10^{-5}$)

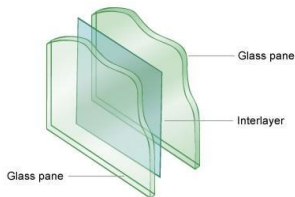


High-contrast layered structures

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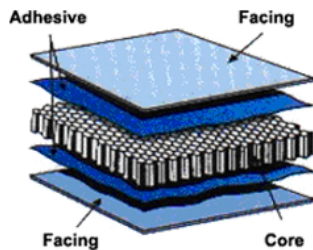
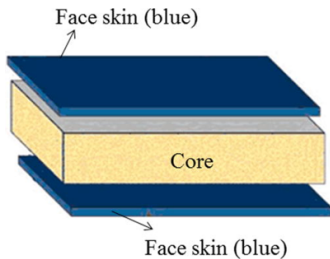


- Laminated glass



Sandwich structures

- Classical sandwich plate



Preliminary remarks

Consider bending of homogeneous isotropic plate of width $2h$

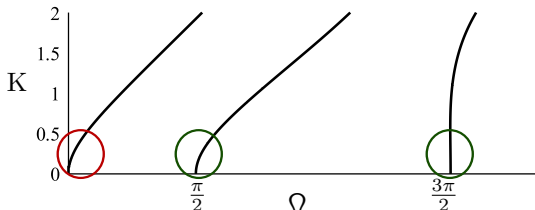
Rayleigh-Lamb dispersion equation

$$\gamma^4 \frac{\sinh \alpha}{\alpha} \cosh \beta - \beta^2 K^2 \cosh \alpha \frac{\sinh \beta}{\beta} = 0, \quad \text{where}$$

$$\alpha^2 = K^2 - \varkappa^2 \Omega^2, \quad \beta^2 = K^2 - \Omega^2, \quad \gamma^2 = K^2 - \frac{1}{2} \Omega^2, \quad \varkappa = \frac{c_2}{c_1}$$

Scaled wave number and frequency

$$K = kh, \quad \Omega = \frac{\omega h}{c_2}$$



Long wave approximations ($K \ll 1$)

○ Low-frequency ($\Omega \ll 1$)

At the leading order $\Omega \sim K^2$ or

$$D_a \frac{d^4 w}{d\xi^4} - \Omega^2 w = 0, \quad \xi = \frac{x}{h}$$

Kirchhoff equation \equiv leading order approximation

○ High-frequency approximations near cut-off frequencies $\Omega_* \sim 1$
($|\Omega - \Omega_*| \ll 1$)

At the leading order $K \sim |\Omega^2 - \Omega_*^2|^{\frac{1}{2}}$

$$P_a \frac{d^2 w}{d\xi^2} + (\Omega^2 - \Omega_*^2) w = 0$$

No overlap frequency regions \Rightarrow

No chance for 2-mode uniform approximations!

Composite (non-uniformly asymptotic) plate theories

Originate from Timoshenko-Reissner-Mindlin ad hoc theories.

$$\overbrace{D_a \frac{d^4 W}{d\xi^4} - \Omega^2 W}^{\text{low-frequency}} + \underbrace{B_a \Omega^2 \frac{d^2 W}{d\xi^2} + C_a \Omega^4 W}_{\text{high-frequency}} = 0,$$

B_a, C_a, D_a - constants

V.L. Berdichevsky. Variational principles of continuum mechanics: I. Fundamentals, 2009

K.C. Le. Vibrations of shells and rods, 2012

I.V. Andrianov et al, Asymptotical mechanics of thin-walled structures, 2013

Contents

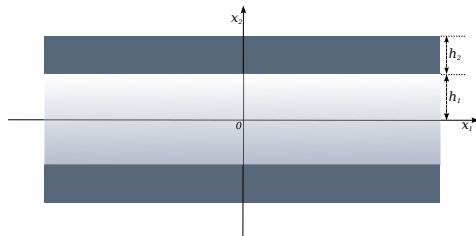
In-plane vector problem

Anti-plane scalar problem

Anti-plane scalar problem for asymmetric plates

In-plane vibrations of three-layered plates

Statement of the problem



Equations of motion

$$\sigma_{ji,j}^q = \rho_q \ddot{u}_i^q, \quad q = c, s \quad \text{for core and skin layers}$$

Boundary and continuity conditions

$$\sigma_{12}^s = 0, \quad \sigma_{22}^s = 0 \quad \text{at} \quad x_2 = h_c + h_s$$

$$\sigma_{12}^c = \sigma_{12}^s, \quad \sigma_{22}^c = \sigma_{22}^s \quad \text{and} \quad u_1^c = u_1^s, \quad u_2^c = u_2^s \quad \text{at} \quad x_2 = h_c$$

Dispersion relation for antisymmetric motion

$$\begin{aligned}
 & 4K^2 h^3 \alpha_2 \beta_2 F_4 [F_1 F_2 C_{\beta_1} S_{\alpha_1} - 2\alpha_1 \beta_1 (\mu - 1) F_3 C_{\alpha_1} S_{\beta_1}] \\
 & + h \alpha_2 \beta_2 C_{\alpha_2} C_{\beta_2} [4\alpha_1 \beta_1 K^2 (h^4 F_3^2 + F_4^2 (\mu - 1)^2) C_{\alpha_1} S_{\beta_1} \\
 & \quad - (4K^4 h^4 F_2^2 + F_4^2 F_1^2) S_{\alpha_1} C_{\beta_1}] \\
 & + C_{\beta_2} S_{\alpha_2} \mu \beta_2 (\beta_2^2 - K^2 h^2) (\beta_1^2 - K^2) [4\alpha_2^2 \beta_1 K^2 h^2 S_{\alpha_1} S_{\beta_1} - F_4^2 \alpha_1 C_{\alpha_1} C_{\beta_1} \\
 & + C_{\alpha_2} S_{\beta_2} \mu \alpha_2 (\beta_2^2 - K^2 h^2) (\beta_1^2 - K^2) [4\alpha_1 \beta_2^2 K^2 h^2 C_{\alpha_1} C_{\beta_1} - F_4^2 \beta_1 S_{\alpha_1} S_{\beta_1} \\
 & + h^3 S_{\alpha_2} S_{\beta_2} [(4\alpha_2^2 \beta_2^2 K^2 F_1^2 + K^2 F_4^2 F_2^2) C_{\beta_1} S_{\alpha_1} \\
 & \quad - \alpha_1 \beta_1 (16\alpha_2^2 \beta_2^2 (\mu - 1)^2 K^4 + F_4^2 F_3^2) C_{\alpha_1} S_{\beta_1}] = 0
 \end{aligned}$$

Non-dimensional scaled frequency and wave number

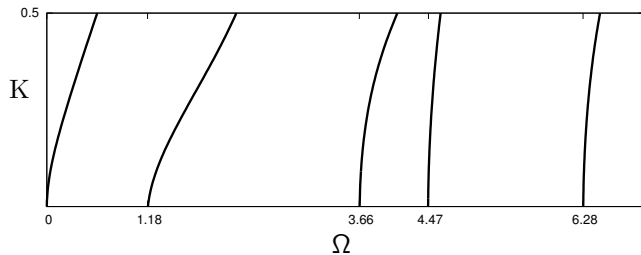
$$\Omega = \frac{\omega h_c}{c_2^c}, \quad K = k h_c$$

$F_i, \quad i = 1..4, \quad \alpha_j, \beta_j, \quad j = 1, 2$ - functions of Ω and K ,

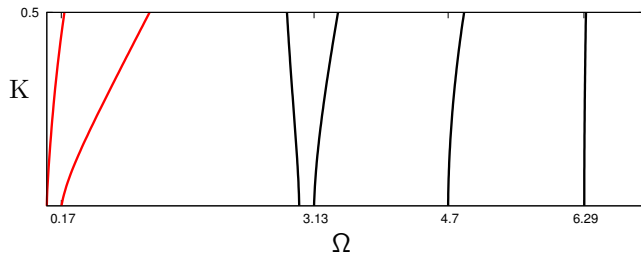
$C_{\alpha_j}, C_{\beta_j}, S_{\alpha_j}, S_{\beta_j}$ - hyperbolic functions

Dispersion curves

No contrast



Effect of contrast



1D eigenvalue problem for shear cut-off (antisymmetric motion)

Setting $\partial u_1^q / \partial x_1 = u_2^q = 0$ in the above problem we have ($u_1^q = u_1^q(x_2)$)



Equations of motion

$$\frac{d^2 u_1^q}{d x_2^2} + \frac{\omega^2}{c_2^q} u_1^q = 0, \quad q = c, s,$$

subject to the boundary and continuity conditions

$$\begin{aligned} \frac{d u_1^s}{d x_2} &= 0 \quad \text{at} \quad x_2 = \pm(h_c + h_s), \\ \mu_c \frac{d u_1^c}{d x_2} &= \mu_s \frac{d u_1^s}{d x_2}, \quad u_1^c = u_1^s \quad \text{at} \quad x_2 = \pm h_c. \end{aligned}$$

Cut-off shear frequencies

Equation for cut-off shear frequencies:

$$\tan(\Omega) \tan\left(\sqrt{\frac{\mu}{\rho}} h \Omega\right) = \sqrt{\mu\rho}$$

Condition for a first shear cut-off frequency to be small

$$\rho \ll h \ll \mu^{-1}, \quad \Omega \approx \left(\frac{\rho}{h}\right)^{1/2},$$

where

$$\mu = \frac{\mu_c}{\mu_s}, \quad \rho = \frac{\rho_c}{\rho_s}, \quad h = \frac{h_s}{h_c}.$$

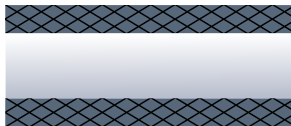
J. Kaplunov et al, Journal of Sound and Vibration, 2016.

Practical examples ($\rho \ll h \ll \mu^{-1}$)

A. Photovoltaic panels

$$\mu \ll 1, h \sim 1, \rho \sim \mu$$

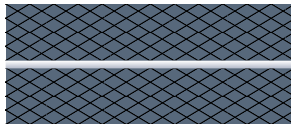
stiff skin layers and light core layer



B. Laminated glass

$$\mu \ll 1, h \sim \mu^{-1/2}, \rho \sim \mu$$

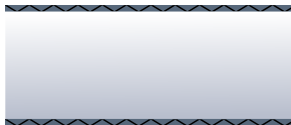
stiff skin layers and light thin core layer



C. Sandwich structures

$$\mu \ll 1, h \sim \mu, \rho \sim \mu^2$$

stiff thin skin layers and light core layer



Unusually low first shear cut-off frequencies!

Long-wave low-frequency asymptotic approximation

For $K \ll 1$ and $\Omega \ll 1$

$$\gamma_1 \Omega^2 + \gamma_2 K^4 + \gamma_3 K^2 \Omega^2 + \gamma_4 K^6 + \gamma_5 \Omega^4 + \gamma_6 K^4 \Omega^2 + \gamma_7 K^8 + \\ \gamma_8 K^2 \Omega^4 + \gamma_9 K^2 \Omega^6 + \gamma_{10} \Omega^6 + \dots = 0$$

Multi-parametric analysis

$$\mu \ll 1, \quad h \sim \mu^a, \quad \rho \sim \mu^b$$

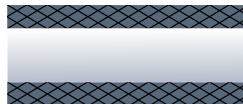
Expanding coefficients

$$\gamma_i \rightarrow G_i \mu^c, \quad G_i \sim 1$$

A. Photovoltaic panels

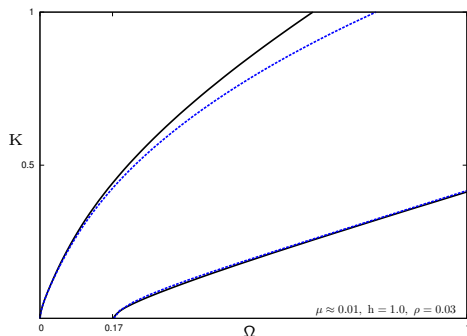
Plate with stiff outer layers and light core

$$\mu \ll 1, \quad h \sim 1, \quad \rho \sim \mu$$



Retain leading order terms for both modes:

1. fundamental mode ($\Omega \sim K^2$)
2. shear mode with cut-off $\Omega_{\text{sh}} \sim \sqrt{\mu}$

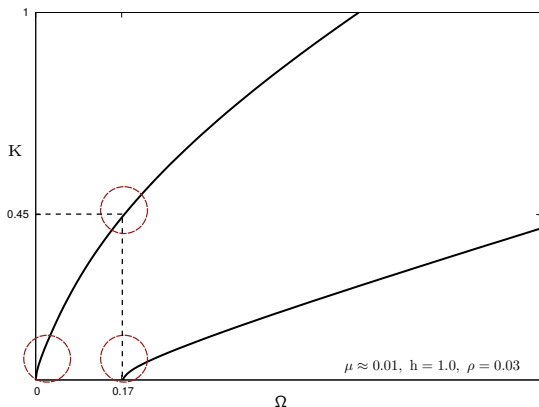


Five term two-mode approximation

$$G_1 \mu \Omega^2 + G_2 \mu K^4 + G_3 K^2 \Omega^2 + G_4 K^6 + G_5 \Omega^4 = 0$$

Local approximations

Three local approximations can be obtained from the two-mode approximation



Local approximations for the fundamental mode

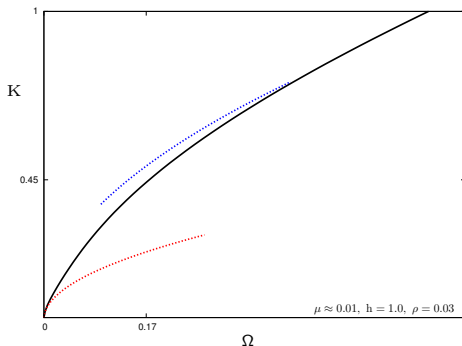
In the vicinity of zero frequency

$$G_1\Omega^2 + G_2K^4 = 0, \quad 0 < K \ll \sqrt{\mu}, \quad \Omega \ll \mu$$

At higher frequencies, including the vicinity of shear cut-off

$$\Omega_{\text{sh}} \sim \sqrt{\mu}$$

$$G_3\Omega^2 + G_4K^4 = 0, \quad \sqrt{\mu} \ll K \ll 1, \quad \mu \ll \Omega \ll 1$$

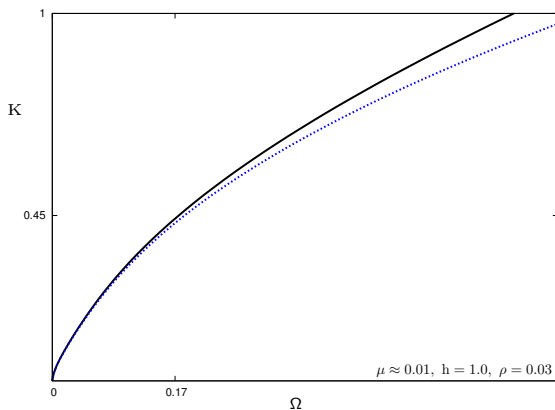


Kirchhoff theory already does not work at $\Omega \sim \mu$!

Uniform approximation for the fundamental mode

Taking both local approximations we derive a uniform one:

$$G_1\mu\Omega^2 + G_2\mu K^4 + G_3K^2\Omega^2 + G_4K^6 = 0$$

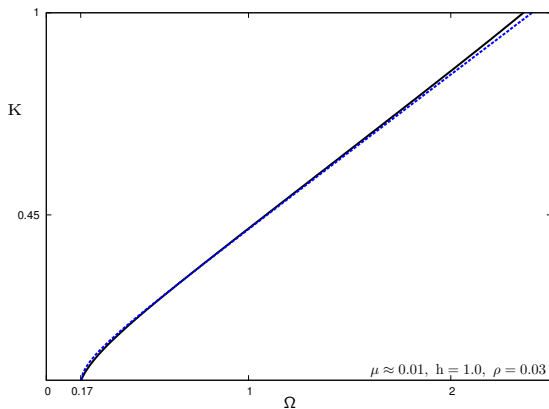


Also valid in the transition region $\Omega \sim \mu, K \sim \sqrt{\mu}$

Near cut-off approximation

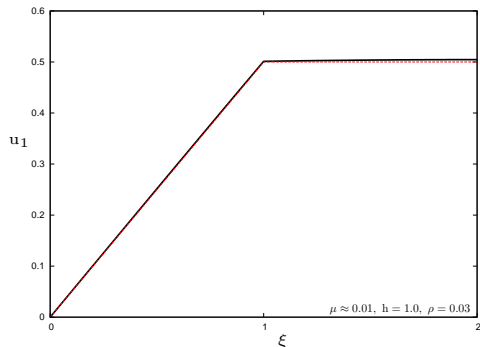
For $\Omega \sim \sqrt{\mu}$, $K \ll 1$

$$G_1\mu + G_3K^2 + G_5\Omega^2 = 0$$



Displacement u_1 near shear cut-off frequency

Horizontal displacement u_1 and approximation for $\mu \approx 0.01$ in shear mode ($K = 0$)



At the leading order

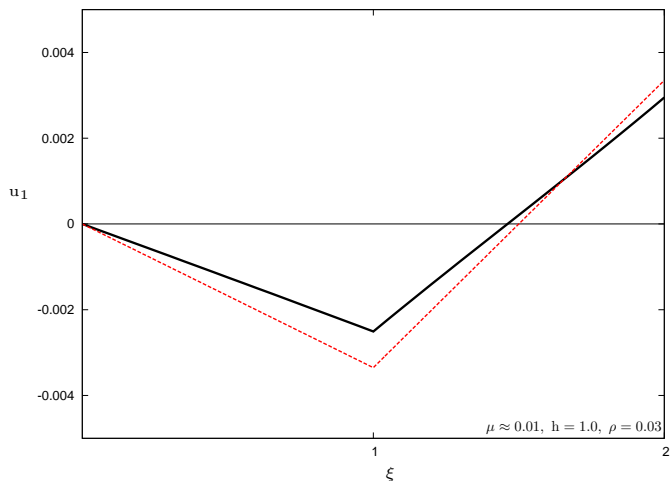
$$u_1 = \begin{cases} \frac{\xi}{1+h}, & \text{for } \xi \leq 1 \\ \frac{1}{1+h}, & \text{for } \xi > 1 \end{cases}$$

where

$$\xi = \frac{x_1}{h_c}$$

Displacement u_1 near shear cut-off frequency

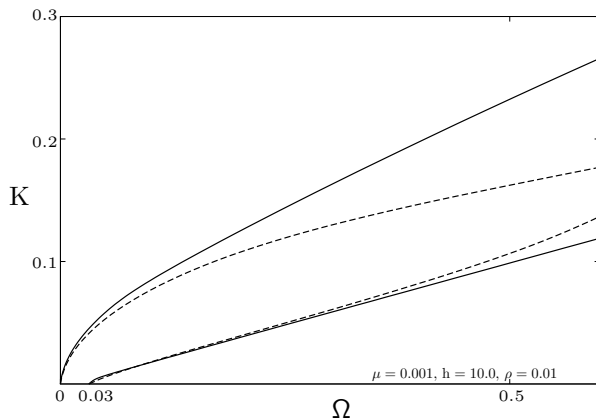
Displacement u_1 and approximation for $\mu \approx 0.01$ on fundamental mode (at $K = 0.45$)



B. Laminated glass. Two-mode approximation

$\Omega_{\text{sh}} \sim \mu^{1/2}$ leads to the two-mode uniform approximation

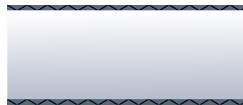
$$G_1 \mu^3 \Omega^2 + G_2 \mu^{3/2} K^4 + G_3 \mu^{3/2} K^2 \Omega^2 + G_4 K^6 + G_5 \mu^2 \Omega^4 = 0,$$



C. Sandwich structure. Two-mode approximation

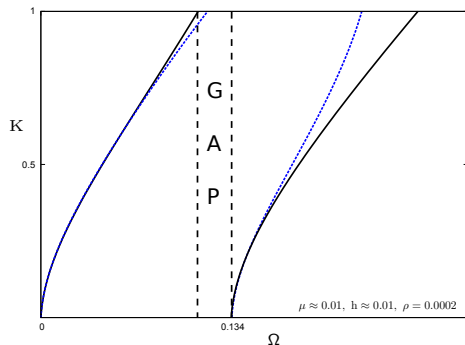
Plate with stiff outer layers and light core

$$\mu \ll 1, \quad h \sim \mu, \quad \rho \sim \mu^2$$



Local approximation:

fundamental mode ($\Omega \ll \sqrt{\mu}$) and shear mode ($\Omega_{\text{sh}} \sim \sqrt{\mu}$)



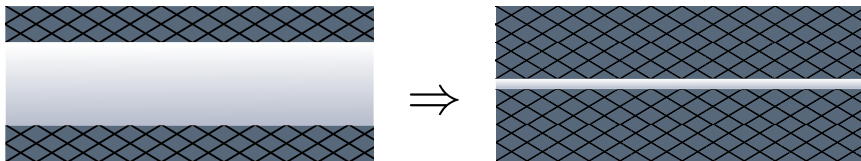
Two-mode approximation

$$\begin{aligned} &G_1 \mu \Omega^2 + G_2 \mu^2 K^4 \\ &+ \mu K^2 \Omega^2 \left(G_3 + \frac{\rho_\mu}{h_\mu} G_8 \right) \\ &+ G_5 \Omega^4 = 0 \end{aligned}$$

Composite non-uniform approximations!

Transition from a uniform to non-uniform approximation

Where is transition from uniform approximation to a composite (non-uniform) one?

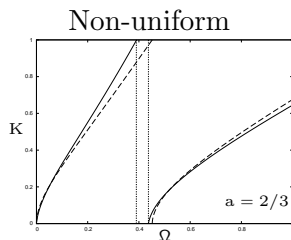
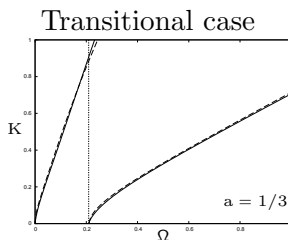
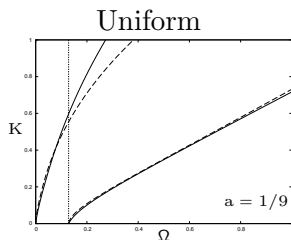


$$\rho = \frac{\rho_c}{\rho_s} \ll 1, \quad \mu = \frac{\mu_c}{\mu_s} \sim \rho, \quad h = \frac{h_s}{h_c} \sim \rho^a, \quad 0 \leq a < 1$$

Small thickness shear cut-off frequency

$$\Omega_{\text{sh}} \approx \left(\frac{\rho}{h} \right)^{1/2} \sim \rho^{(1-a)/2} \ll 1$$

Transition from a uniform to non-uniform approximation



- Uniform $0 \leq a \leq 1/3$

$$\rho^{1-a}G_1\Omega^2 + \rho^{1-a}G_2K^4 + G_3K^2\Omega^2 + \frac{1}{3}\rho^{2a}G_2K^6 + G_5\Omega^4 = 0.$$

- Non-uniform $1/3 < a < 1$

$$\rho^{1-a}G_1\Omega^2 + \rho^{1-a}G_2K^4 + G_3K^2\Omega^2 + G_5\Omega^4 = 0.$$

In progress: 2D PDEs for strongly inhomogeneous plates

Uniformly asymptotic



$$G_1 \mu u_{tt} + G_2 \mu \Delta^2 u + G_3 \Delta u_{tt} + G_4 \Delta^3 u + G_5 u_{tttt} = 0$$

Composite



$$G_1 \mu u_{tt} + G_2 \mu^2 \Delta^2 u + G_3 \mu \Delta u_{tt} + G_5 u_{tttt} + G_8 \Delta u_{tttt} = 0$$

Not easy to justify!

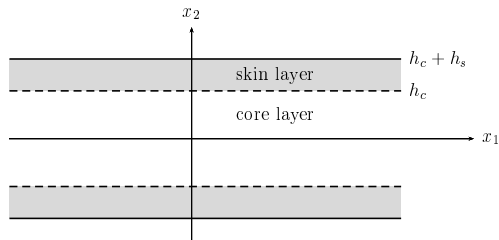
Contents

In-plane vector problem

Anti-plane scalar problem

Anti-plane scalar problem for asymmetric plates

Anti-plane antisymmetric motion



Equations of motion

$$\frac{\partial \sigma_{13}^q}{\partial x_1} + \frac{\partial \sigma_{23}^q}{\partial x_2} - \rho_q \frac{\partial^2 u_q}{\partial t^2} = 0, \quad q = c, s,$$

with

$$\sigma_{i3}^q = \mu_q \frac{\partial u_q}{\partial x_i}, \quad i = 1, 2,$$

u_q are out of plane displacements, σ_{i3}^q are shear stresses.

Dispersion relation

Continuity conditions along interfaces $x_2 = \pm h_c$

$$\sigma_{23}^c = \sigma_{23}^s \quad \text{and} \quad u_c = u_s.$$

Traction-free boundary conditions

$$\sigma_{23}^s = 0 \quad \text{at} \quad x_2 = \pm(h_c + h_s).$$

Equations of motion

$$\Delta u_q - \frac{1}{(c_2^q)^2} \frac{\partial^2 u_q}{\partial t^2} = 0, \quad q = c, s.$$

Dispersion relation

$$\mu \alpha_1 \cosh(\alpha_1) \cosh(\alpha_2 h) + \alpha_2 \sinh(\alpha_1) \sinh(\alpha_2 h) = 0,$$

with

$$\alpha_1 = \sqrt{K^2 - \Omega^2}, \quad \alpha_2 = \sqrt{K^2 - \frac{\mu}{\rho} \Omega^2},$$
$$\Omega = \frac{\omega h_c}{c_2^c}, \quad K = k h_c, \quad h = \frac{h_s}{h_c}, \quad \mu = \frac{\mu_c}{\mu_s}, \quad \rho = \frac{\rho_c}{\rho_s}.$$

Exact solutions for displacements and stresses

$$u_c = h_c \frac{\sinh(\alpha_1 \xi_{2c})}{\alpha_1}, \quad \sigma_{13}^c = i\mu_c K \frac{\sinh(\alpha_1 \xi_{2c})}{\alpha_1}, \quad \sigma_{23}^c = \mu_c \cosh(\alpha_1 \xi_{2c}),$$

and

$$\begin{aligned} u_s &= h_c \beta (\cosh [\alpha_2 (h \xi_{2s} + 1)] - \tanh [\alpha_2 (h + 1)] \sinh [\alpha_2 (h \xi_{2s} + 1)]), \\ \sigma_{13}^s &= i\mu_s K \beta (\cosh [\alpha_2 (h \xi_{2s} + 1)] - \tanh [\alpha_2 (h + 1)] \sinh [\alpha_2 (h \xi_{2s} + 1)]), \\ \sigma_{23}^s &= \mu_s \alpha_2 \beta (\sinh [\alpha_2 (h \xi_{2s} + 1)] - \tanh [\alpha_2 (h + 1)] \cosh [\alpha_2 (h \xi_{2s} + 1)]), \end{aligned}$$

where

$$\beta = \frac{\sinh \alpha_1}{\alpha_1 (\cosh \alpha_2 - \sinh \alpha_2 \tanh [\alpha_2 (h + 1)])}.$$

Dimensionless variables

$$\begin{aligned} \xi_{2c} &= \frac{x_2}{h_c}, & 0 \leq x_2 \leq h_c, \\ \xi_{2s} &= \frac{x_2 - h_c}{h_s}, & h_c \leq x_2 \leq h_c + h_s. \end{aligned}$$

Long-wave low-frequency limit

Polynomial dispersion relation

$$\mu + \gamma_1 K^2 + \gamma_2 K^4 + \gamma_3 K^2 \Omega^2 + \gamma_4 \Omega^2 + \gamma_5 \Omega^4 + \dots = 0,$$

with

$$\gamma_1 = \frac{\mu}{2} (1 + h^2) + h,$$

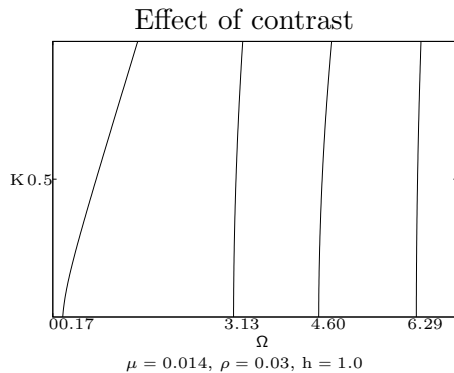
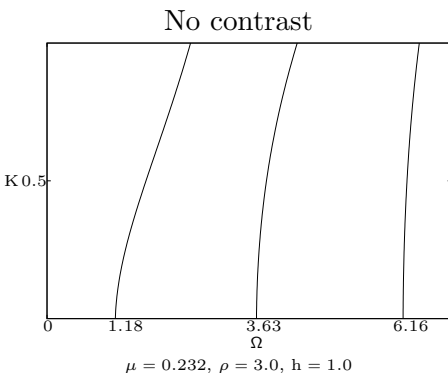
$$\gamma_2 = \frac{\mu}{24} (1 + 6h^2 + h^4) + \frac{h}{6} (1 + h^2),$$

$$\gamma_3 = -\frac{\mu}{12} (1 + 3h^2) - \frac{h}{6} - \frac{\mu h}{12\rho} (2 + 3\mu h) - \frac{\mu h^3}{12\rho} (4 + \mu h),$$

$$\gamma_4 = -\frac{\mu}{2} - \frac{\mu h}{\rho} \left(1 + \frac{\mu h}{2} \right),$$

$$\gamma_5 = \frac{\mu}{24} + \frac{\mu h}{12\rho} (2 + 3\mu h) + \frac{\mu^2 h^3}{24\rho^2} (4 + \mu h).$$

Dispersion curves



- No fundamental mode. It appears in case of symmetric motion.
- The lowest cut-off frequency in case of a contrast is $\Omega = 0.17$

Consider two setups of the contrast:

A. Photovoltaic panels and B. Sandwich structures

A. Photovoltaic panels. Shortened polynomial dispersion relation

Plate with stiff outer layers and light core

$$\mu \ll 1, \quad h \sim 1, \quad \rho \sim \mu$$



$$\gamma_1 \sim \gamma_2 \sim \gamma_3 \sim \gamma_4 \sim \gamma_5 \sim 1.$$

Shortened dispersion relation

$$\frac{\mu}{h} + K^2 - \frac{1}{\rho_\mu} \Omega^2 = 0.$$

Scaled dimensionless frequency and wavenumber

$$\Omega^2 = \mu^\alpha \Omega_*^2 \quad \text{and} \quad K^2 = \mu^\alpha K_*^2,$$

where $\Omega_* \sim K_* \sim 1$ and $0 < \alpha \leq 1$.

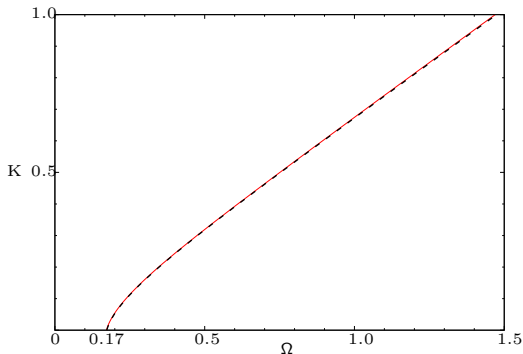
α covers the whole long-wave low-frequency band, given by $\Omega \ll 1$, and $K \ll 1$.

Shortened polynomial dispersion relation

Dispersion relation expressed in Ω_* and K_* becomes

$$\Omega_*^2 = \rho_\mu \left(K_*^2 + \frac{\mu^{1-\alpha}}{h} \right).$$

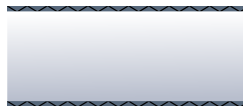
At $\alpha < 1$ we have $\Omega_* \sim \sqrt{\rho_\mu} K_*$ or $\omega \sim c_2^s k$, corresponding to the short-wave limit for stiffer skin layers.



B. Sandwich structure. Shortened polynomial dispersion relation

Plate with stiff outer layers and light core

$$\mu \ll 1, \quad h \sim \mu, \quad \rho \sim \mu^2$$



$$\gamma_1 \sim \gamma_2 \sim \mu \quad \text{and} \quad \gamma_3 \sim \gamma_4 \sim \gamma_5 \sim 1.$$

Approximate dispersion relation

$$\mu + \mu \left(\frac{1}{2} + h_\mu \right) K^2 - \frac{h_\mu}{6\rho_\mu} K^2 \Omega^2 - \left(\frac{\mu}{2} + \frac{h_\mu}{\rho_\mu} \right) \Omega^2 + \frac{h_\mu}{6\rho_\mu} \Omega^4 = 0.$$

Normalized wavenumber and frequency

$$K^2 = \mu K_*^2 \quad \text{and} \quad \Omega^2 = \mu \Omega_*^2,$$

we obtain

$$1 + \mu \left(\frac{1}{2} + h_\mu \right) K_*^2 - \mu \frac{h_\mu}{6\rho_\mu} K_*^2 \Omega_*^2 - \left(\frac{\mu}{2} + \frac{h_\mu}{\rho_\mu} \right) \Omega_*^2 + \mu \frac{h_\mu}{6\rho_\mu} \Omega_*^4 = 0.$$

Shortened polynomial dispersion relation

Adapt a near cut-off asymptotic expansion in the form

$$\Omega_*^2 = \Omega_0^2 + \mu \Omega_1^2 + \dots$$

where

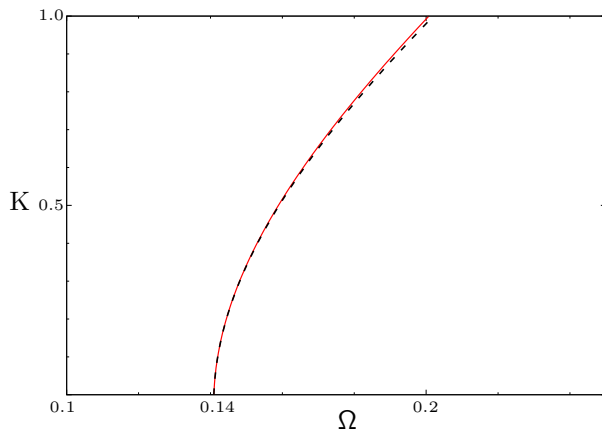
$$\Omega_0^2 = \frac{\rho_\mu}{h_\mu} \quad \text{and} \quad \Omega_1^2 = \frac{\rho_\mu}{h_\mu} \left(\frac{1}{3} + h_\mu \right) K_*^2 - \frac{1}{3} \frac{\rho_\mu^2}{h_\mu^2},$$

leading to the optimal shortened dispersion relation

$$\left(h_\mu + \frac{1}{3} \right) K^2 - \frac{1}{\mu} \frac{h_\mu}{\rho_\mu} \Omega^2 + \left(1 - \frac{\mu \rho_\mu}{3 h_\mu} \right) = 0.$$

Valid only over a narrow vicinity of the cut-off frequency!

Numerical illustration



$\mu = 0.014$, $\rho = 0.03$, and $h = 1.0$

Asymptotic formulae for displacements and stresses (setup A)

Leading order displacements and stresses

$$\begin{aligned}u_c &= h_c \xi_{2c}, \\ \sigma_{13}^c &= i \mu_c \sqrt{\mu} K_* \xi_{2c}, \\ \sigma_{23}^c &= \mu_c,\end{aligned}$$

and

$$\begin{aligned}u_s &= h_c, \\ \sigma_{13}^s &= i \mu_s \sqrt{\mu} K_*, \\ \sigma_{23}^s &= \mu_c h \left(K_*^2 - \frac{\Omega_*^2}{\rho_\mu} \right) (\xi_{2s} - 1).\end{aligned}$$

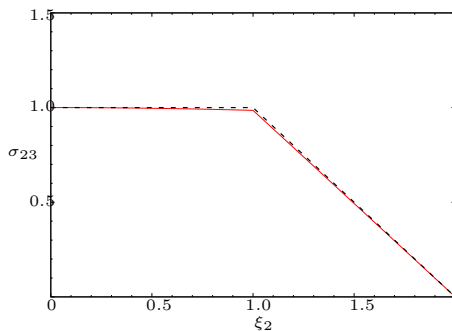
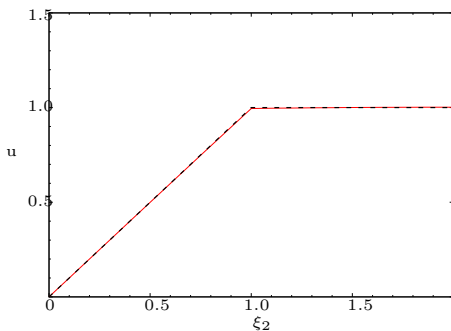
We obtain

$$\frac{u_q}{h_c} \sim \frac{\sigma_{23}^q}{\mu_c} \sim \frac{\sigma_{13}^q}{\mu_q \sqrt{\mu}}, \quad q = c, s.$$

Normalised displacement and stress σ_{23} (setup A)

$$\xi_2 = \xi_{2c}, u = \frac{u_c}{h_c}, \text{ and } \sigma_{23} = \frac{\sigma_{23}^c}{\mu_c}, (0 < \xi_2 \leq 1)$$

$$\text{or } \xi_2 = 1 + \xi_{2s}, u = \frac{u_s}{h_c}, \text{ and } \sigma_{23} = \frac{\sigma_{23}^s}{\mu_c}, (1 < \xi_2 \leq 2)$$



Model construction (setup A)

Scaled longitudinal coordinate and time

$$x_1 = \frac{h_c}{\sqrt{\mu}} \xi_1 \quad \text{and} \quad t = \frac{h_c}{c_{2c} \sqrt{\mu}} \tau,$$

Normalised displacement and stresses

$$u^q = h_c v^q, \quad \sigma_{13}^q = \mu_q \sqrt{\mu} S_{13}^q, \quad \sigma_{23}^q = \mu_c S_{23}^q, \quad q = c, s.$$

with all dimensionless quantities assumed to be of order unity.

Core layer

Skin layer

$$\begin{aligned} \mu \frac{\partial S_{13}^c}{\partial \xi_1} + \frac{\partial S_{23}^c}{\partial \xi_{2c}} - \mu \frac{\partial^2 v^c}{\partial \tau^2} &= 0, \\ S_{13}^c &= \frac{\partial v^c}{\partial \xi_1}, \quad S_{23}^c = \frac{\partial v^c}{\partial \xi_{2c}}. \end{aligned}$$

$$\begin{aligned} \frac{\partial S_{13}^s}{\partial \xi_1} + \frac{1}{h} \frac{\partial S_{23}^s}{\partial \xi_{2s}} - \frac{1}{\rho_\mu} \frac{\partial^2 v^s}{\partial \tau^2} &= 0, \\ S_{13}^s &= \frac{\partial v^s}{\partial \xi_1}, \quad \mu h S_{23}^s = \frac{\partial v^s}{\partial \xi_{2s}}. \end{aligned}$$

Derivation of a shortened equation (setup A)

Continuity and boundary conditions

$$\begin{aligned} v^c|_{\xi_{2c}=1} &= v^s|_{\xi_{2s}=0} , \\ S_{23}^c|_{\xi_{2c}=1} &= S_{23}^s|_{\xi_{2s}=0} , \end{aligned}$$

and

$$S_{23}^s|_{\xi_{2s}=1} = 0.$$

Expand displacements and stresses into asymptotic series as

$$\begin{aligned} v^q &= v_0^q + \mu v_1^q + \cdots , \\ S_{j3}^q &= S_{j3,0}^q + \mu S_{j3,1}^q + \cdots , \quad q = c, s \quad \text{and} \quad j = 1, 2. \end{aligned}$$

Leading order problem

$$S_{13,0}^c = \frac{\partial v_0^c}{\partial \xi_1}, \quad \frac{\partial S_{23,0}^c}{\partial \xi_{2c}} = 0, \quad S_{23,0}^c = \frac{\partial v_0^c}{\partial \xi_{2c}},$$

and

$$\frac{\partial S_{13,0}^s}{\partial \xi_1} + \frac{1}{h} \frac{\partial S_{23,0}^s}{\partial \xi_{2s}} - \frac{1}{\rho_\mu} \frac{\partial^2 v_0^s}{\partial \tau^2} = 0,$$

$$S_{13,0}^s = \frac{\partial v_0^s}{\partial \xi_1}, \quad \frac{\partial v_0^s}{\partial \xi_{2s}} = 0,$$

with

$$\begin{aligned} v_0^c \Big|_{\xi_{2c}=1} &= v_0^s \Big|_{\xi_{2s}=0}, \\ S_{23,0}^c \Big|_{\xi_{2c}=1} &= S_{23,0}^s \Big|_{\xi_{2s}=0}, \end{aligned}$$

and

$$S_{23}^s \Big|_{\xi_{2s}=1} = 0.$$

Leading order solution

$$v_0^s = w(\xi_1, \tau).$$

The rest of the quantities are expressed in terms of w as

$$\begin{aligned} S_{13,0}^c &= \xi_{2c} \frac{\partial w}{\partial \xi_1}, & S_{23,0}^c &= w, & v_0^c &= \xi_{2c} w, \\ S_{13,0}^s &= \frac{\partial w}{\partial \xi_1}, & S_{23,0}^s &= w(1 - \xi_{2s}), \end{aligned}$$

with w satisfying the 1D equation

$$\frac{\partial^2 w}{\partial \xi_1^2} - \frac{1}{\rho_\mu} \frac{\partial^2 w}{\partial \tau^2} - \frac{1}{h} w = 0,$$

which may be presented in the original variables as

$$\frac{\partial^2 u_s}{\partial x_1^2} - \frac{\rho_s}{\mu_s} \frac{\partial^2 u_s}{\partial t^2} - \frac{\mu_c}{\mu_s h_c h_s} u_s = 0,$$

where $u_s(x_1, t) \approx w(x_1, t)$.

Justification of the model

Insert ansatz $u_s = \exp \{i(kx_1 - \omega t)\}$ into the last equation. As a result, we have the dispersion relation

$$k^2 - \frac{\rho_s}{\mu_s} \omega^2 + \frac{\mu_c}{\mu_s h_c h_s} = 0.$$

Coincides with the shortened dispersion relation for setup A!

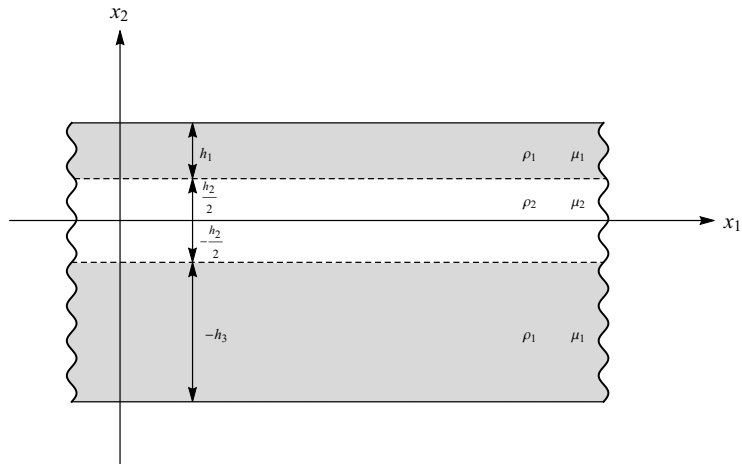
Contents

In-plane vector problem

Anti-plane scalar problem

Anti-plane scalar problem for asymmetric plates

Anti-plane shear of three-layered asymmetric plates



More sophisticated dispersion relation

$$\mu\alpha_1\alpha_2 \tanh(h\alpha_1) + \mu^2\alpha_2^2 \tanh(\alpha_2) + \\ \mu\alpha_1\alpha_2 \tanh(h^*\alpha_1) + \alpha_1^2 \tanh(h^*\alpha_1) \tanh(\alpha_2) \tanh(h\alpha_1) = 0,$$

where

$$\alpha_1 = \sqrt{K^2 - \frac{\mu}{\rho}\Omega^2}, \quad \alpha_2 = \sqrt{K^2 - \Omega^2},$$

with

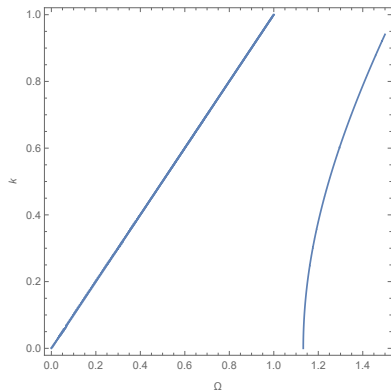
$$\Omega = \frac{\omega h_2}{c_2^{(2)}}, \quad K = kh_2,$$

and

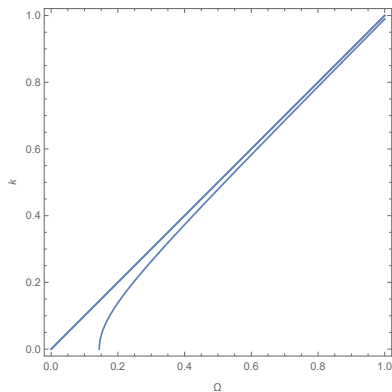
$$h = \frac{h_1}{h_2}, \quad h^* = \frac{h_3}{h_2}, \quad \mu = \frac{\mu_2}{\mu_1}, \quad \rho = \frac{\rho_2}{\rho_1}, \quad c_2^{(i)} = \sqrt{\frac{\mu_i}{\rho_i}}, \quad i = 1, 2$$

Effect of contrast

No contrast



Contrast parameters



Two modes in case of high contrast for a scalar problem!

Cut-off frequencies

Frequency equation

$$\begin{aligned} & \sqrt{\mu\rho} \left(\tan \left(h \sqrt{\frac{\mu}{\rho}} \Omega \right) + \tan \left(h^* \sqrt{\frac{\mu}{\rho}} \Omega \right) \right) \\ & + \mu\rho \tan(\Omega) - \tan \left(h \sqrt{\frac{\mu}{\rho}} \Omega \right) \tan(\Omega) \tan \left(h^* \sqrt{\frac{\mu}{\rho}} \Omega \right) = 0. \end{aligned}$$

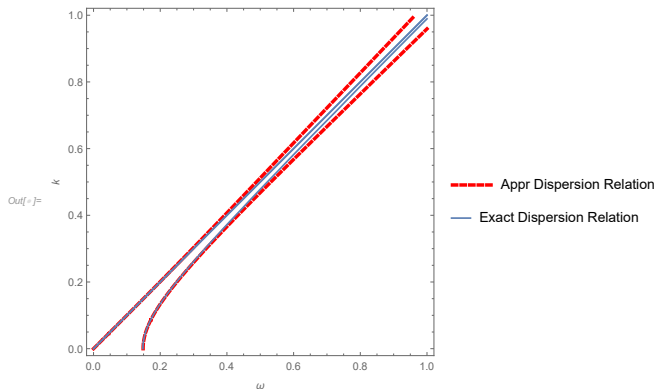
Lowest cut-off

$$\Omega \approx \sqrt{\frac{\mu\rho(h + h^* + \rho)}{hh^*\mu}}$$

A. Photovoltaic panels. Two-mode approximation

Shortened polynomial dispersion relation for two modes

$$G_1 K^2 + G_2 \Omega^2 + G_3 K^4 + G_4 K^2 \Omega^2 + G_5 \Omega^4 + G_6 K^4 \Omega^2 + G_7 K^2 \Omega^4 = 0$$



Concluding remarks

- Multi-parametric analysis is performed
- One- and two-mode approximations (both asymptotically uniform and composite) are constructed
- 1D shortened PDEs are derived for several setups.